

Noise Generation & Attenuation



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- 1. General
- 2. Jet fans
- 3. Axial Fans

General



Sound Power L_W depends on

Vibrations

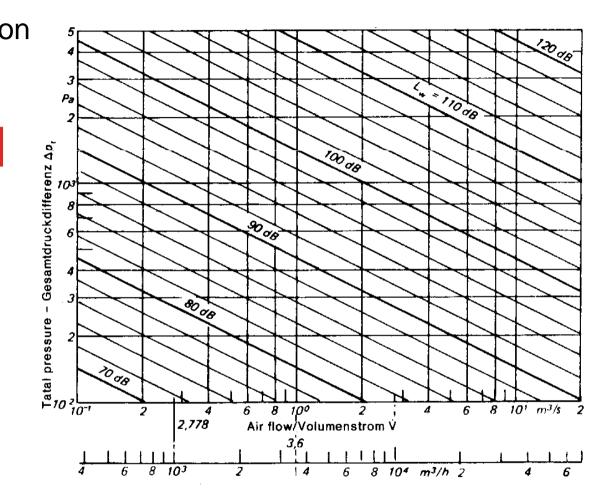
Number of blades



Turbulences

Rotational speed

Power consumption



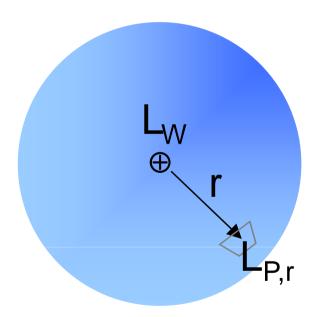
Sound power as a function of fan capacity and pressure at $\eta_{\text{max.}}$



Sound Pressure L_{P.r}:

depends on distance r from source L_W

- Think of a soap bubble (constant amount of soap)
 The larger the bubble becomes (r) ...
 the thinner the thickness of the sphere becomes
- → The thickness of the sphere is proportional with L_{P,r}





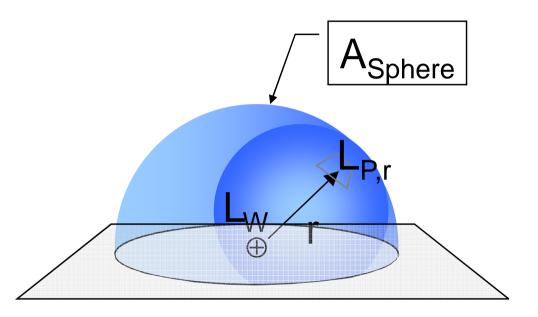
Sound Pressure L_{P.r}:

$$L_{P,r} = L_{W} - L_{A_{Sphere}}$$

$$L_{P,r} = L_{W} - 10 \cdot \log(A_{Sphere})$$

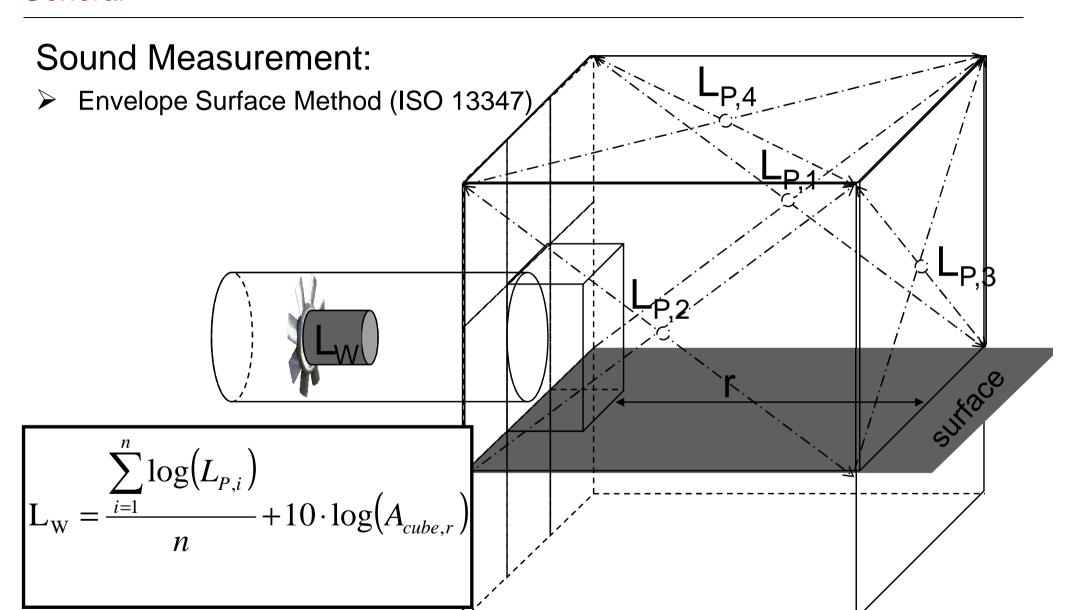
$$A_{Sphere} = 4 \cdot \pi \cdot r^2$$

$$A_{\text{Semi-sphere}} = 2 \cdot \pi \cdot r^2$$



General





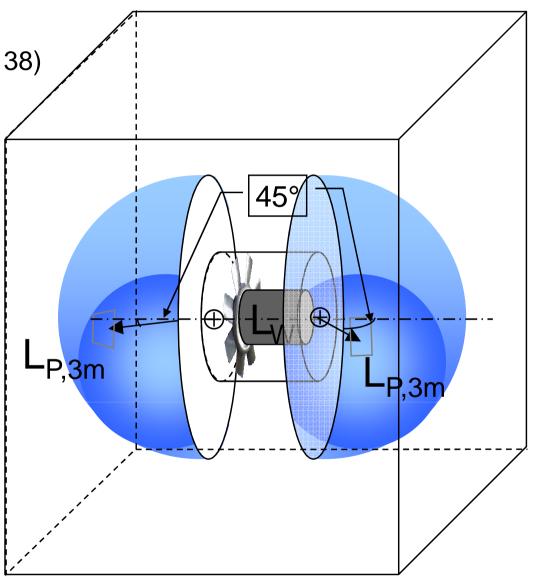
General



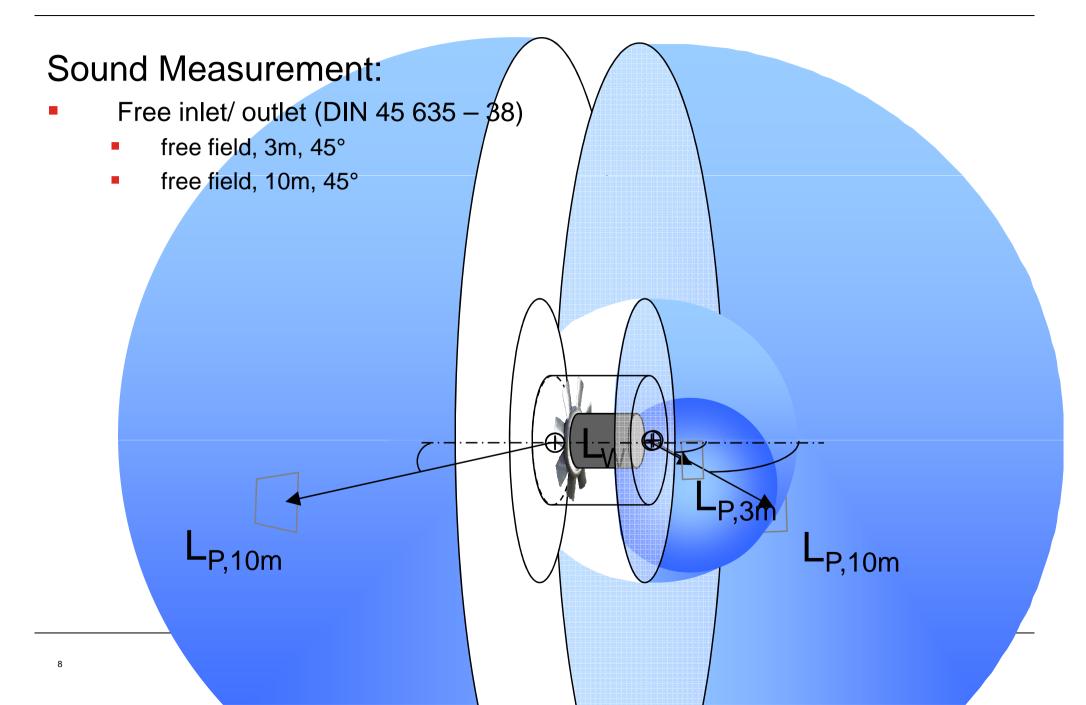
Sound Measurement:

Free inlet/ outlet (DIN 45 635 − 38)

> free field, 3m, 45°



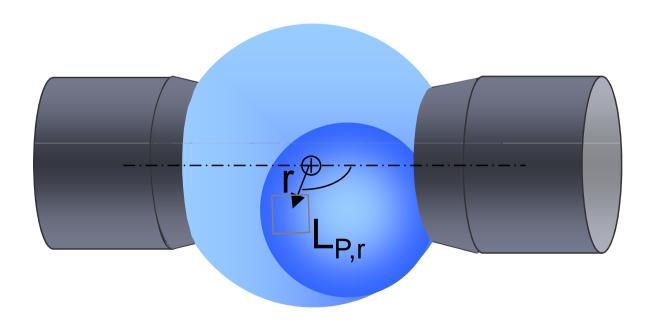






Sound Measurement:

- Free inlet/ outlet (DIN 45 635 − 38)
 - free field, 3m, 45°
- Casing break-out Noise
 - free field, 1m, 90°



С



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Batteries of Parallel Jet Fans

No. of Parallel Fans	$\Delta L_{P\!\!,\;install}$ [dB]
2	+3
3	+5
4	+6



> from "free field (3m, 45°) " to "tunnel"

Tunnel Wall Type	$\Delta L_{P, tunnel}$ [dB]
Tiled Walls	Approx +6
Concrete Walls	Approx. +2.5
Blasted Rock	±0



$$L_{P,free field, 3m, 45^{\circ}} = L_{P,tunnel} - \Delta L_{P, install} - \Delta L_{P, tunnel}$$

Sample:

 $L_{P,free field, 3m, 45^{\circ}} = 85dB(A) - 6dB - 2.5dB$

 $L_{P,free field, 3m, 45^{\circ}} \sim 76.5 dB(A)$

→ used value for jet fan selection



- Protection Grills = Bird Screens
 - approx. + 2dB(A)
 (tip: place grill in silencer at 50% of length)
 - approx. 2 % losses for each grill on top of shaft power



	Length of Silencers	$\Delta L_{P\!,\;silencer}$ [dB]
0.5D		approx2.0
1D		approx6.0
1.5D		approx8.5
2D		approx11.0
3D		approx16.0

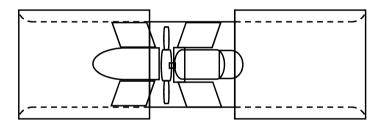
longer silencers not really reasonable



Silencers Types:

WITT&SOHN

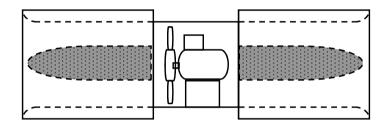
(without pod)



Advantages WITT&SOHN:

- No additional losses
- Low Sound pressure level due to sophisticated design
- High efficiency fan

Competitors use (with pod)



Disadvantages with pod:

- Fan efficiency decreased
- Costs increased due to additional pod



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- by round silencers (see jet fans above)
- by split silencers:

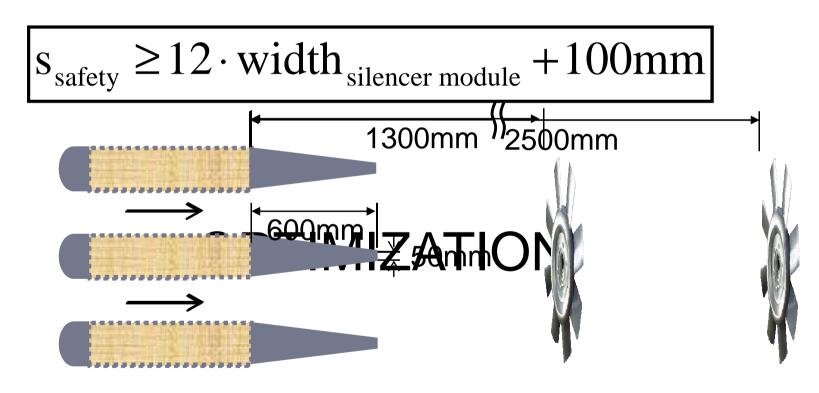
$$\Delta p_{loss} \sim \frac{\text{width}_{silencer module}}{\text{width}_{opening}} \sim 100 - 200 \text{Pa}$$

- max. air speed inside silencer:

$$c_{air} \le 15 \,\text{m/s}$$



- by split silencers:
 - min. distances before impeller:

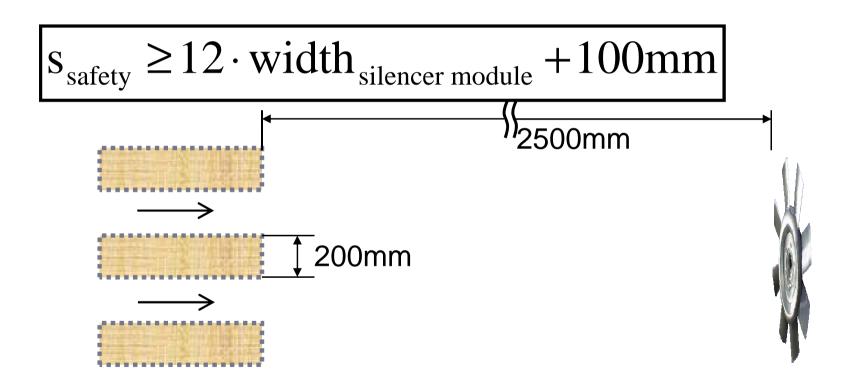




otherwise stall operation likely!

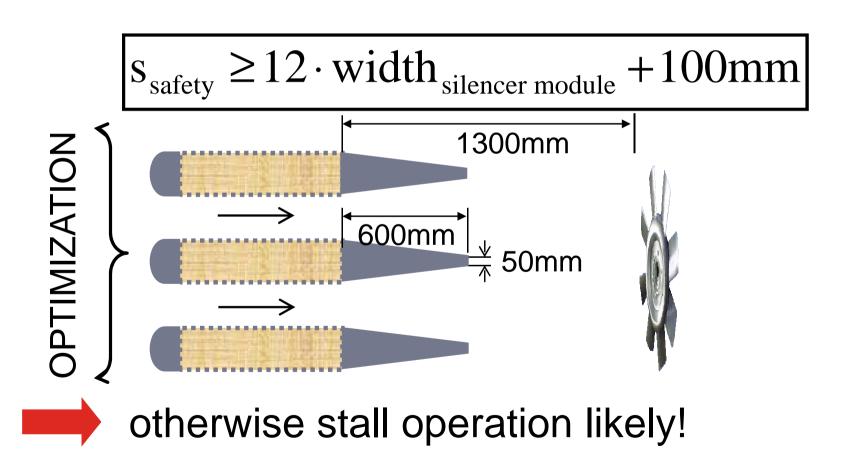


- by split silencers:
 - min. distances before impeller:





- by split silencers:
 - min. distances before impeller:



Summary



- The sound power level of a fan is mainly determined by it's volume flow rate / pressure characteristic
- Sound pressure can be measured, sound power is calculated
- Influence of number of jet fans installed in tunnel on sound pressure level
- Influence of tunnel wall types on sound pressure level
- How much attenuation you can achieve with increasing length of silencers for jet fans
- Disadvantages of silencers with pods for jet fans
- Silencers for axial fans